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# **INVECOR pretest calculations of molten pool and lower head model**

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# **Objectives**

- Determine characteristics of molten pool, temperature and stress-and-strain conditions of the lower head model
- Optimize conditions of melt heating and develop recommendations on the location of electrodes and model geometry

# Methodology

- Multi-optional calculations of molten pool thermal hydrodynamics at different model geometries and electrode positioning using the DYMELT program:
- Integral approach was used for solving problems of melt hydrodynamics taking into account melting/crystallization and thermal conductivity of the solid wall
- 3-D formulation (1/10 of the lower head model sector 36°)
- Quasi-stationary Laminar regime of convection (Navier-Stoks system of equations, energy and continuity)
- Calculations of the model thermal conditions using the ANSYS package

# Basic model of the vessel Characteristics of the model

- Semi-elliptical lower head
- Inner diameter 400 mm
- Wall thickness 90 mm
- Number of heaters 5
- Minimum distance from the heater bottom to the internal surface of the model– 10 mm
- Distance from the heater axis to the model axis 120 mm
- Heater diameter 90 mm
- Height of the heater active zone 45 mm
- Height of the heater passive zone 10 mm
- Distance from the top point of the heater active zone to corium surface 5 mm
- Distance from corium surface to thermal screen 25 mm
- Power capacity of a single heater 18 kW, total power – 90 kW
- Corium mass 64 kg



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## **Basic model of the vessel(2)**

### **Boundary conditions**

- External water-cooled surface of the lower head model – 100°C
- Heat flux from corium top surface by radiation through transparent gas to screen and lateral surface of the model
- Ideal screen zero heat flux from its outside surface
- Heating part of electrodes is assumed to have a uniform heat flux distribution
- Not heating part of electrodes has thermal insulation
- There is no upward heat sink along electrodes

### **Basic model of the vessel(3)**

**Thermophysical parameters** 

- Solidus temperature 2150°C (corium C-30)
- Corium liquidus temperature 2300°C
- Thermal conductivity of solid and liquid corium 3.3 W/m·K
- Thermal conductivity of steel 30 W/m·K
- Thermal conductivity of corium and steel does not depend on temperature
- Emissivity of corium, inside surface of the model and screen 0.8

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### **Basic model of the vessel(4)**

Molten pool configuration **Calculation results** Upward surface of molten pool Downward surface of molten pool

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## **Basic model of the vessel (5)**

#### **Corium temperature field in the axial section of the heater**



### **Basic model of the vessel (6)**

### Corium temperature field in the section rotated by 18°



## **Basic model of the vessel (7)**

### **Corium temperature field in the inter- heater cross-section**



### **Basic model of the vessel(8)**

#### **Calculation results** Crust thickness in the axial section of the heater



### **Basic model of the vessel(9)**

#### **Calculation results**

### Crust thickness in the section rotated by 18°



### **Basic model of the vessel(10)**



### **Basic model of the vessel(11)**

### **Calculation results**

Temperature field in the lower head model

Max temperature does not exceed ≈700°C



of the lower head model is necessary



## **Basic model of the vessel (12)**

### **Calculation with external thermal insulation**

- The wall temperature condition was calculated by using ANSYS
- •Boundary condition:
- heat flux distribution on the internal wall produced by molten pool modeling
- 100 C at water cooled external wall Thermal insulation of axi-symmetrical belt on the external surface of the lower head model and of the surface above the screen
- Thermal insulation is ideal

## **Basic model of the vessel (13)**

### **Calculation results**

Temperature field in the lower head model with partial thermal insulation



Maximum temperature about 1200°C

# Basic model of the vessel (14) Calculation with electrodes positioned in one plane (quasi-slice geometry)

 ✓ In order to get realistic temperatures and heat fluxes on the internal vessel surface, at least near the axial plane of the model, a version with all electrodes positioned in the plane was considered

 ✓ For that model diameter was increased to 500 mm, and the mass of corium charge to 150 kg. Note: for this a part of solid charge can be placed on the model periphery, the rest of mass can be pored from the furnace

 ✓ Other characteristics of the model and boundary conditions are close to the basic version





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### **Basic model of the vessel (15)**

**Calculation results** 

### Molten pool configuration



#### ✓ Substantial improvement has not been achieved

## **Basic model of the vessel(16)**

### **Temperature distribution in vertical sections**



## **Basic model of the vessel (17)**

### Main results of the basic model calculations

- Bottom part of the basic model cannot be efficiently heated
- Molten pool is small, considerable part of solid corium lies under it. Elliptical part of the lower head "does not work"
- In absence of external insulation temperature on the internal surface of the model does not exceed ≈700°C
- A considerable thickness of the model wall and consequent heat fluxes along the wall require the insulation of a large surface. At this maximum temperature on the internal surface of the model is approx. 1200°C
- If the wall thickness is reduced, smaller surface can be insulated, but in this case the pool is much smaller

## **Cylindrical model**

### **Model characteristics**

- Internal diameter of the model 300 mm
- Wall thickness 90 mm
- Number of heaters 5
- Minimum distance from the heater bottom to the model internal surface– 10 mm
- Distance from the heater axis to the model axis 75 mm
- Heater diameter 70 mm
- Height of the heater active part – 90 mm
- Height of the heater lower (passive) part – 20 mm
- Distance from the top point of the active part to corium surface 45 mm
- Distance from the corium surface to screen – 25 mm
- Power of a single heater 18kW, Total power – 90 kW
- Corium mass 64 kg





# Cylindrical model(2)

Calculation methodology, boundary conditions and thermophysical parameters are the same as in the basic model (calculations without partial insulation)

## Cylindrical model(3)



## Cylindrical model(4)

### **Calculation results** Temperature field in the axial section of the heater



## Cylindrical model(5)

### **Calculation results** Crust thickness in the axial section of the heater



### Cylindrical model(6)

#### Temperature field if the model wall thickness is 80 mm



# Cylindrical model(7)

### Main results of alternative model calculations

- Small azimuthal divergence of calculated parameters: crust thickness, distribution of temperatures and heat fluxes, etc. in comparison with the basic model
- Fraction of molten corium is larger in comparison with the basic model
- This model geometry enables to use electrodes having a longer heating part than in the basic version
- High temperature of the internal surface in the vessel top even in absence of external insulation. A certain wall superheating above the melting temperature can be avoided by the wall thickness reduction

Note: Model is easier to manufacture

## Conclusions

- Due to the inherent limits on the power and number of electrodes the semi-elliptical model is not suitable for the preparation of a realistic pool
- Partial thermal insulation of the external wall enables to raise temperature on the internal surface to the necessary level but does not influence the molten pool configuration and volume
- Quasi-slice geometry does not provide the cardinal solution of the existing problems
- Of the considered options the cylindrical model can be recommended